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2013 Regional Conference

Weld Sizing:

Using ACT for Robust Customization



Fluid Dynamics

Structural Mechanics

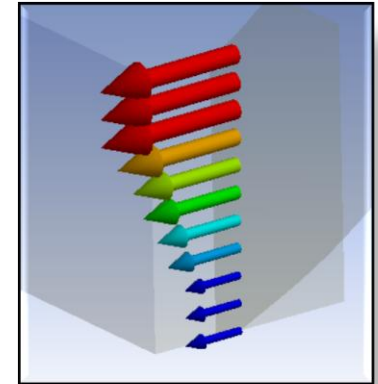
Electromagnetics

Systems and Multiphysics

Rod Scholl Epsilon FEA, LLC

Weld Sizing: Using ACT for Robust Customization

1. Weld Sizing – Historical Context
2. Customization – From APDL to ACT
3. The Weld Extension Details
 - Geometric Intelligence
 - Determining Required Weld Size
 - Plotting Results
 - Level of Effort



Weld Sizing: Historical Context

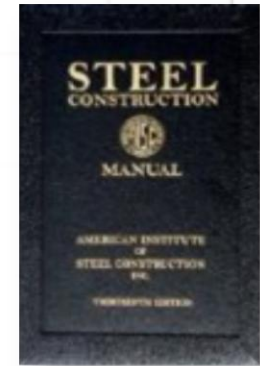
Weld Sizing often controlled by standards:

AISC (American Institute of Steel Construction)

AWS (American Welding Society)

ASME Boiler Pressure Vessel Code (BPVC)

IIW (International Institute of Welding)

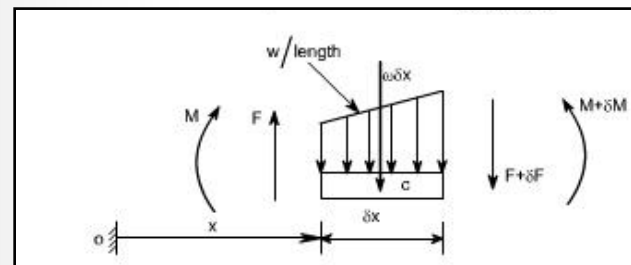
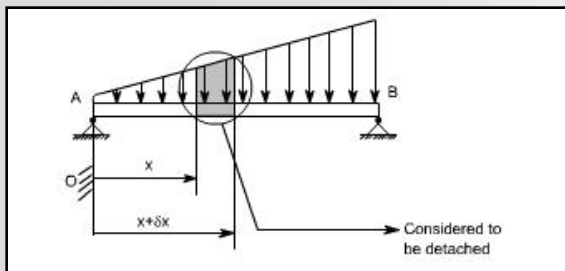


Historically sized with hand-calcs

- Geared toward beams more than irregular geometry
- Commonly uses Loads (Force and Moments)
 - Not stresses



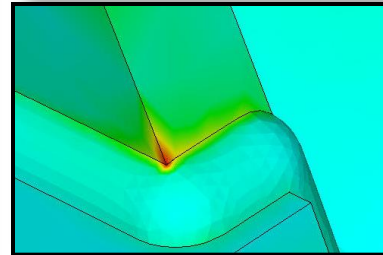
American Welding Society
THE STRENGTH
OF WELDING



FEA Models Increasingly Available

FEA results are too localized

- Singularities & Stress concentrations

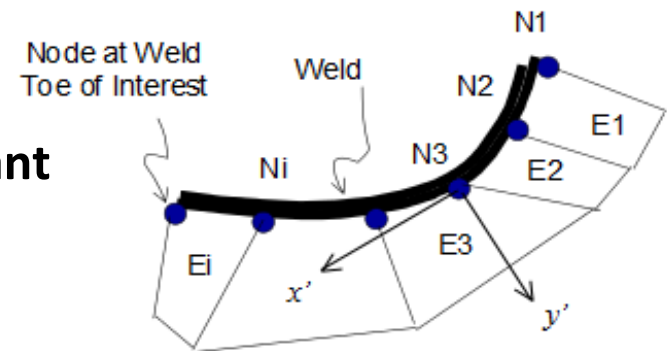


“Hot-spot stress” Methods

1. Expects explicitly modeled size/geometry & requires nodes at toe
 - Must iterate on weld size
 - Some methods expect shell models
2. Geared toward fatigue failure at toe
3. Is predictive, not specification-compliant

Verity Method (Battelle, 2006)

- Integrated into FE-Safe and Hypermesh
- Has the three limitations above
- Is insensitive to mesh density and has better correlation/acceptance



Each company uses different specs, approaches, and methods!

Weld Sizing: Design for Manufacture

Welding Manufacturing Costs are often nontrivial

- Never-ending squeeze on labor/materials
- Cost of FEA continues to fall (relative to build-and-test)
 - High cycle fatigue usually not testable at system level
- Margins added on top of Safety Factors
 - Weld process has huge variability
 - Indeterminate physical material properties
 - Residual stresses
 - Weld length/flaw count adds statistical factor
 - **Statically indeterminate loading (not a simple beam)**
 - *This portion of overdesign can be mitigated using FEA*



Customization: From APDL to ACT

The Power of Customization

- **Cost benefits**
 - Automation usually has large ROI
- **Quality Benefits**
 - Standardization/Codification of Design Process
- **Individual Benefits**
 - Increasing individual contribution over time

The Risks of Customization

- **Obsolescence (ANSYS development always adding features)**
- **Version Repair / Upkeep / Training & Documentation**

Customization: From APDL to ACT

APDL (ANSYS Parametric Design Language)

- Scripting in APDL has been viable bridge (snippets)

Journaling / Scripting in Workbench allows for task automation

- *DO loop functionality for repetitive tasks
- Log File allows a low initial learning investment
- Limited interaction with numerical model/results

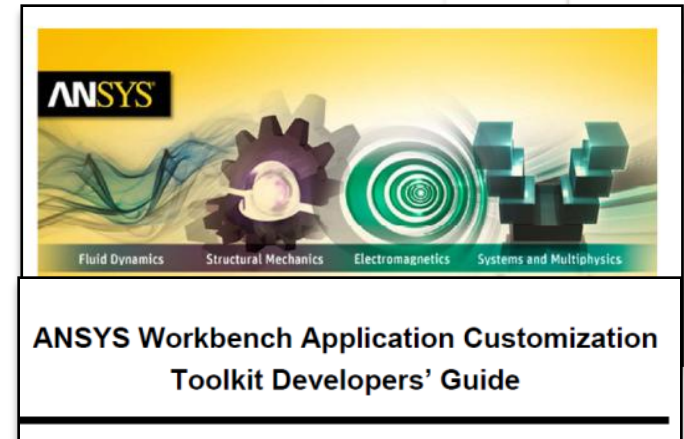
ACT (ANSYS Customization Toolkit) – New!

- Full access and manipulation of the model / matrices via API's
- Can encapsulate legacy APDL scripts with low investment
- Object Oriented! (Python)
- ANSYS inc. committed to long term support
- Expanding Library of Free Extensions (some open source)

Downloads at the customer portal of <http://support.ANSYS.com>

ACT Released at 14.5

- Manageable learning curve
 - *Python* is within many graduates' skill sets
(Yes, we're getting old)
 - Developers' guide from ANSYS
 - Training is available
 - Basic Tool to browse Workbench API's
- Extension can call uncompiled APDL scripts
 - Do what you can in Python -- put the rest in APDL
 - Incrementally learn Python
 - Edit APDL without ACT license!
- Robust Unit inputs
- You get to make your own button icon
...which is fun.



Ansys.AAP.PublicAPIs.Selection Namespace

Declaration Syntax

namespace Ansys.AAP.PublicAPIs.Selection

Table 1: Members

ISelectionInfo	SelectionInfo class
ISelectionMgr	ISelectionMgr Interface
SelectionTypeEnum	SelectionTypeEnum enumerator

ISelectionInfo

SelectionInfo class

Declaration Syntax

public interface ISelectionInfo

Table 2: Members

Property	Id	Gets or sets the ID of this ISelectionInfo object.
Property	Ids	Gets or sets selected IDs.
Property	Name	Gets or sets the name of this ISelectionInfo object.
Property	SelectionType	Gets or sets the selection type.

The Weld Extension

The Weld Extension has four steps

1. Pass surface selection and results to Mechanical APDL

Required because we are including an APDL core

Mechanical will launch Mechanical APDL in background

2. Intelligent Geometry Recognition

3. Load/Stress Calculation

4. Pass results back to WB and Plot

This is the challenging part

Step 1: Pass Selection and Results to Mechanical APDL

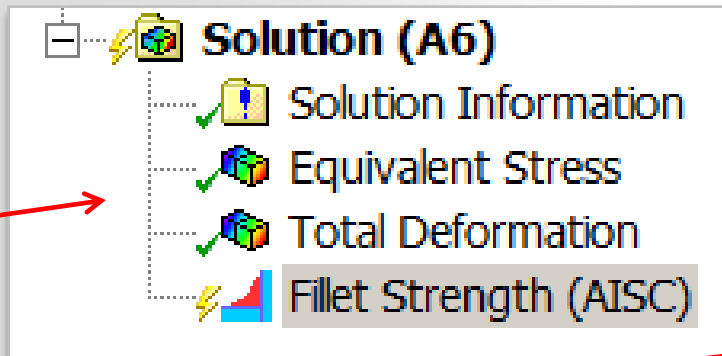
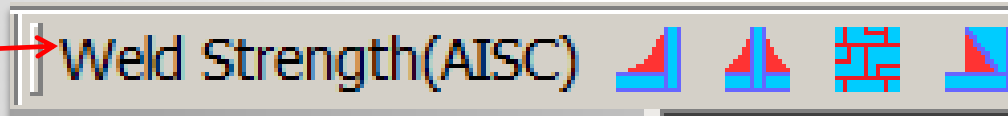
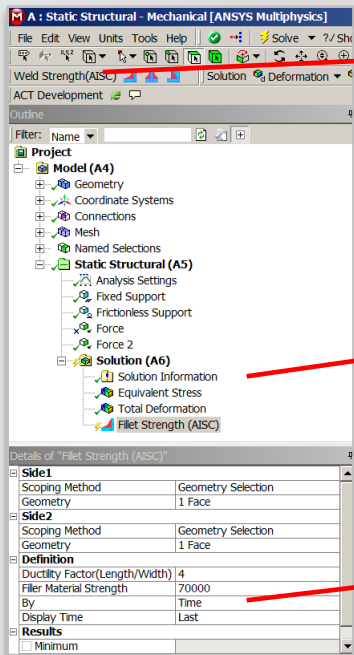
Pass surface selection and results to Mechanical APDL

1. This is simple (once you get the hang of it)

Can enforce scoping to one type(s) of geometry

In our case, it is restricted to two surfaces

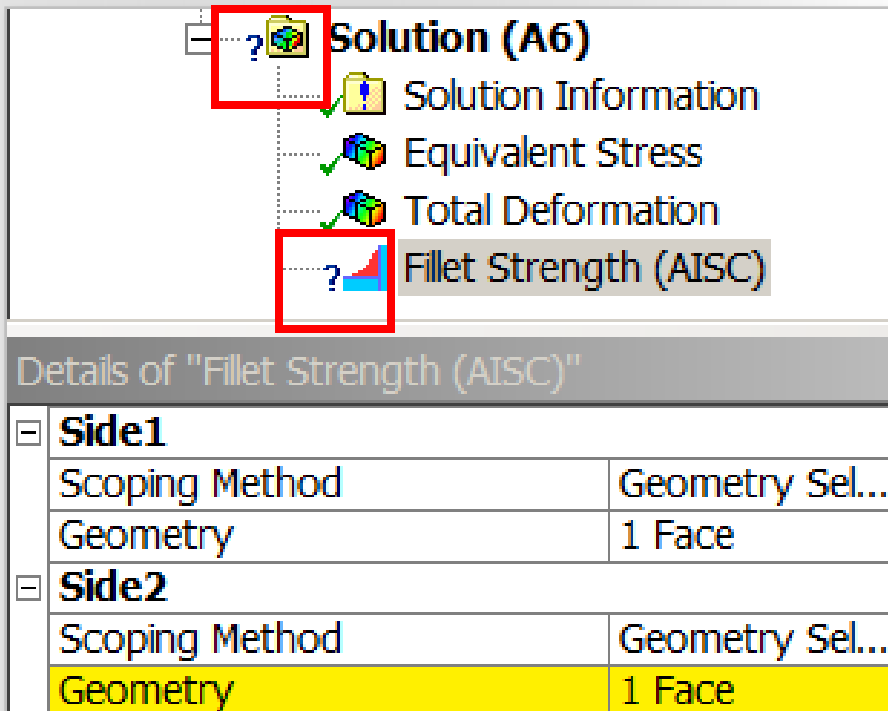
Enter other data in details window (as dropdown or variable)



Side1	
Scoping Method	Geometry Selection
Geometry	1 Face
Side2	
Scoping Method	Geometry Selection
Geometry	1 Face
Definition	
Ductility Factor (Length/Width)	4
Filler Material Strength	70000
By	Time
Display Time	Last

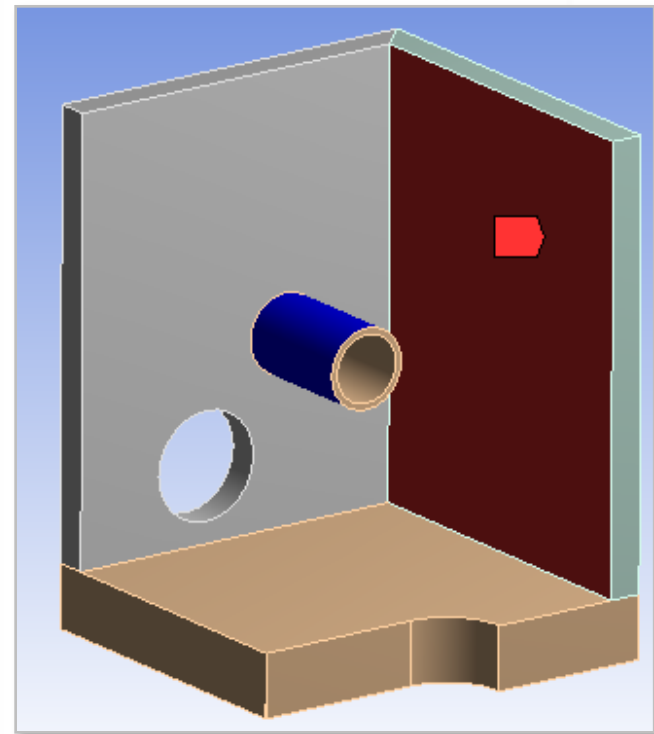
Step 1: Pass Selection and Results, cont'd

Enforce error checking to prevent incompatible choices



The screenshot shows the ANSYS software interface. The Solution (A6) tree is expanded, showing Solution Information, Equivalent Stress, Total Deformation, and Fillet Strength (AISC). The Fillet Strength (AISC) item is highlighted with a red box. Below the tree, the details for "Fillet Strength (AISC)" are shown in a table format.

Details of "Fillet Strength (AISC)"	
[-] Side1	
Scoping Method	Geometry Sel...
Geometry	1 Face
[-] Side2	
Scoping Method	Geometry Sel...
Geometry	1 Face



No Intersection!

Step 1: Pass Selection and Results, cont'd

Plot weld size requirement per AISC Strength

Or plot the weld stresses in local coordinates

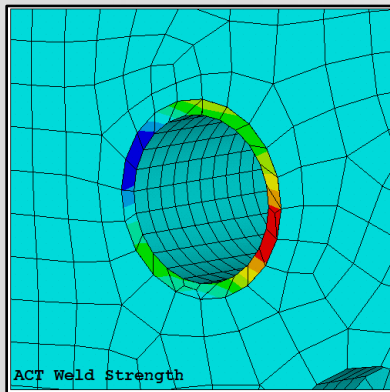
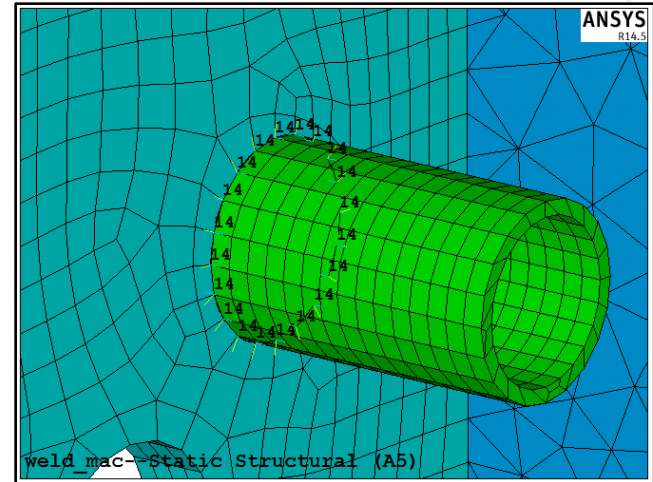
Definition	
Result Type	AISC Strength
Ductility Factor(Length/Width)	AISC Strength
Filler Material Strength	Stress

Definition	
Result Type	Stress
Stress Direction	SX (Avg. Normal)
Ductility Factor(Length/Width)	SX (Avg. Normal)
Filler Material Strength	SYX (Avg. Shear Parallel)
By	SXZ (Avg. Shear Perpendicular)
Display Time	SX (Weld-Side Normal)
Results	
<input type="checkbox"/> Minimum	SYX (Weld-Side Shear Parallel)
	SXZ (Weld-Side Shear Perpendicular)
	SX (Far-Side Normal)
	SYX (Far-Side Shear Parallel)
	SXZ (Far-Side Shear Perpendicular)

Step 2: Geometry Recognition

Robust Geometry Recognition

- Using APDL, determined normal thickness (150 lines of script)
 - Done for each shared node
 - Allows for curved weld joints
 - Allows for varying thickness
- Queries normal and shear stresses
 - On both parent materials
 - On front and back of solid



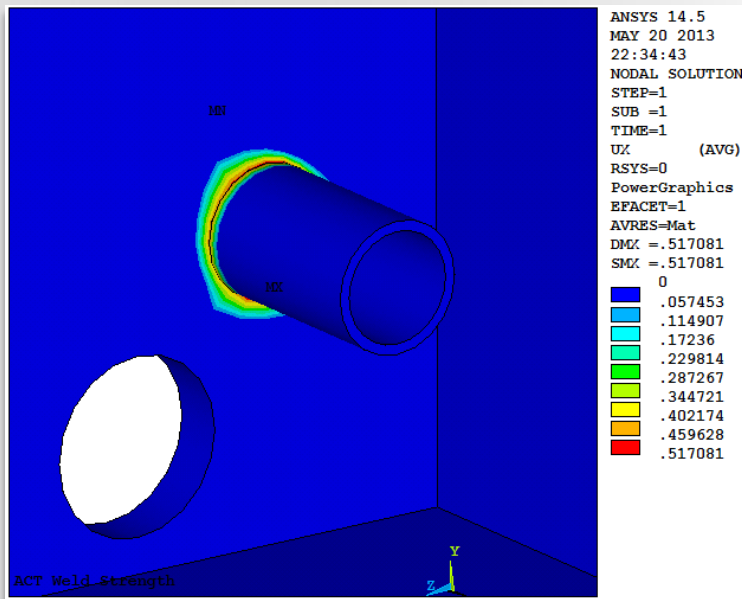
```

*do,jcount,1,ncount
!builds stress array in individual CS's
CS,CS_max,0,storenum(jcount),nxaxisA(jcount),nyaxisA(jcount)
rsys,CS_max
*get,normstress(jcount),node,storenum(jcount),s,y
*get,shearijstress(jcount),node,storenum(jcount),s,xy
*get,shearikstress(jcount),node,storenum(jcount),s,yz
*get,oppnormstress(jcount),node,storeopp(jcount),s,y
*get,oppshearijstress(jcount),node,storeopp(jcount),s,xy
*get,oppshearikstress(jcount),node,storeopp(jcount),s,yz
*enddo
  
```

Step 3: Averaging Stresses

Averaged along length 4X the thickness (100 lines of script)

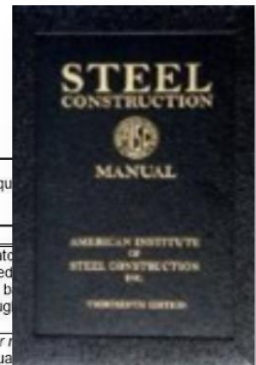
- One of many methods to account for ductility
- Determined minimum Weld Size (25 lines of script)
 - Used AISC method, Chapter J, Table J2.5
 - Different extensions/calc's for each weld type: double fillet, PJP, etc.



Available Strength of Welded Joints

Load Type and Direction Relative to Weld Axis	Pertinent Metal	ϕ and Ω	Nominal Strength (F_u or F_w)	Effective Area (A_{fu} or A_{fv})	Requirements
COMPLETE-JOINT-PENETRATION GROOVE WELDS					
Tension Normal to weld axis			Strength of the joint is controlled by the base metal		Matched with base metal toughness
Compression Normal to weld axis			Strength of the joint is controlled by the base metal		Filler metal equal or less than matching filler metal is permitted.
Tension or Compression Parallel to weld axis			Tension or compression in parts joined parallel to a weld need not be considered in design of welds joining the parts.		Filler metal with a strength level equal to or less than matching filler metal is permitted
Shear			Strength of the joint is controlled by the base metal		Matching filler metal shall be used. ^(c)

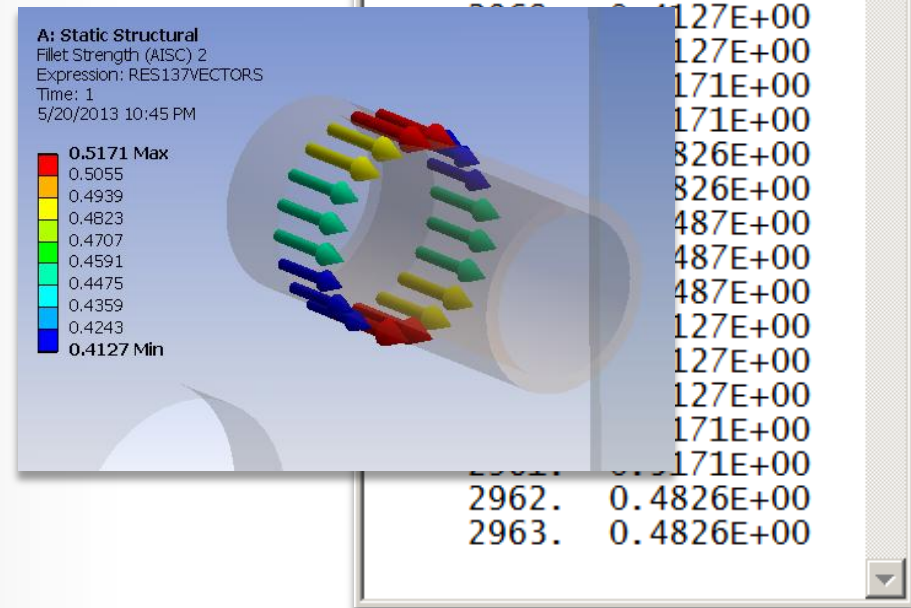
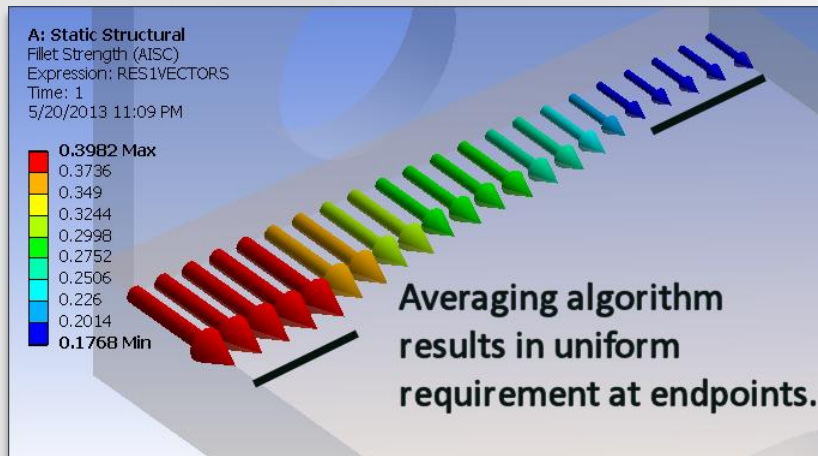
(Part of Table J2.5 AISC 2005)



Step 4: Plotting Results

Pass results into Workbench Mechanical for plotting

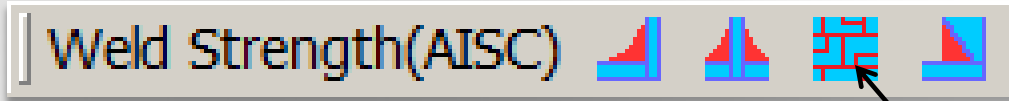
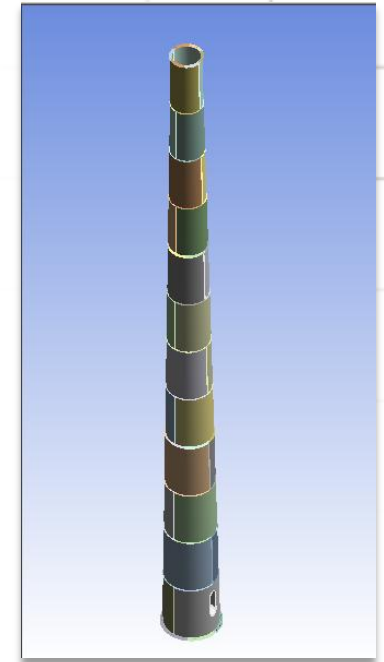
- Uses ascii text file, written from Mechanical APDL
- Read into Mechanical and plotted
 - Vector style plot chosen
 - Option to plot raw stresses (normal , shear, etc.)



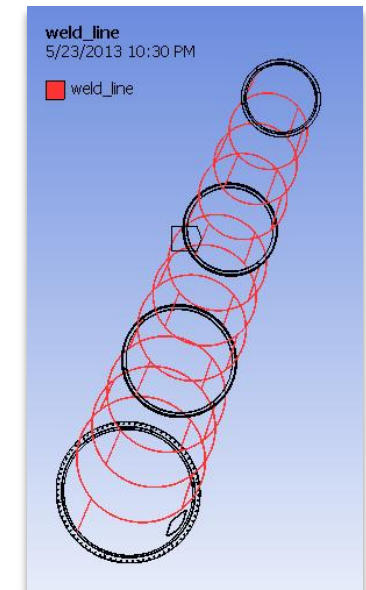
System Level Example: Wind Turbine Tower

Extend to Large Model

- Full model used for modal/vibration/seismic
- CJP welds expense vs. PJP
- Loading
 - Tapered cylindrical bending (blade load/moment)
 - Wind load varies top to bottom (not linearly)
 - Wind load varies circumferentially (drag)
 - Not an easy hand-calc
 - (differential equation might be an unsolved form)

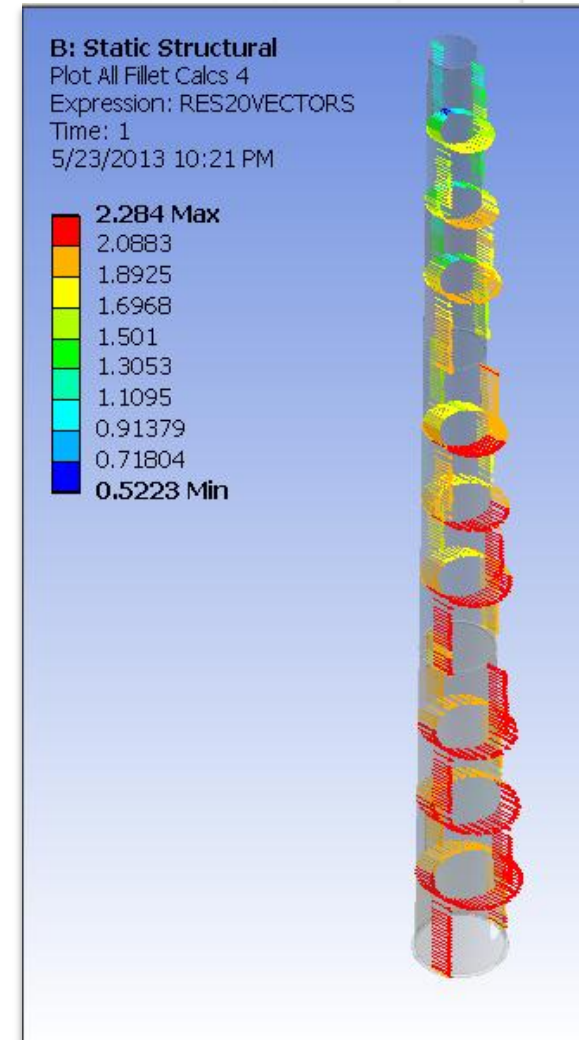
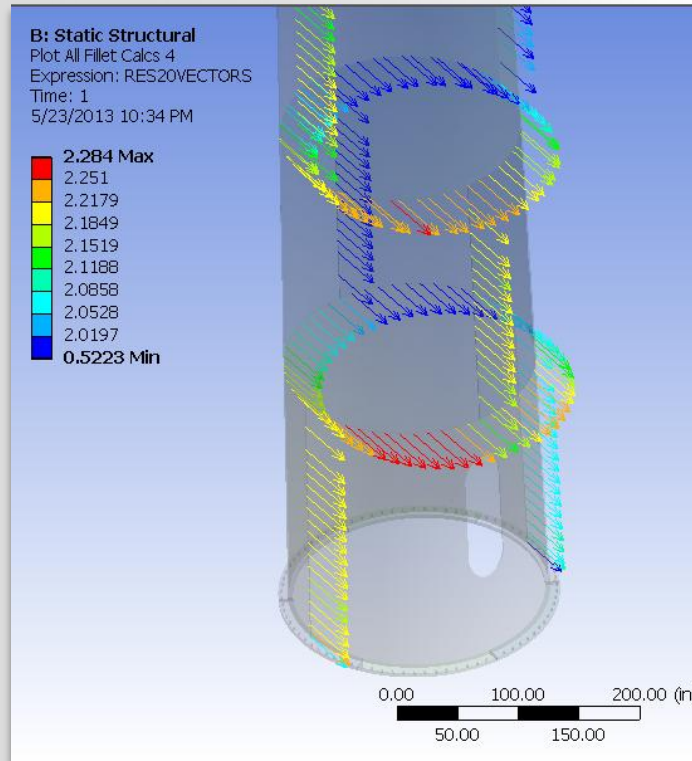


Plot all results (alpha)



System Level Example: Wind Turbine Tower

Material is 2" thick, so everything < 2" may have cost savings from PJP welds.



Level of Effort / Q & A

40 hrs -- Reaching (very) Basic Functionality with ACT

- No previous experience with python
- Used training materials, tutorials, and sample files
- It would've been *much* easier/quicker with a mentor or instructor

3 hrs – Later, wrote “Plot all Welds” Extension

- Not user friendly, but functional for me.

80 hrs – Writing the APDL Script

- Could've doubled or halved that depending on topology flexibility

Q & A