... within Epsilon

Twin Cities
ANSYS® User Meeting

November 2011

Mesh Discretization Error

Mesh Discretization Error

1. Mesh Discretization: The “One-sided” Error Source
2. Tet Pregidous
3. Case Study A: Shape-Functions Effect
   - With and without mid-side nodes
   - Stresses & Deflection
4. Case Study B: Mesh Convergence
   - Node vs. Element (Averaged vs Unaveraged)
   - PRERR (SEPC/SMXB)

Mesh Discretization Error

- Often small compared to load/material property error/scatter
- Ownership of error lands on analyst
  - Often linked to “credibility” of whole analysis
- True Error analysis would likely show Mesh Discretization is minor issue
  - And yet… Scrutiny continues
  - And rules and criteria abound… (while other scatter goes unmentioned)

“It can be *measured*? Well let’s fixate on it!”

Mesh Discretization Error

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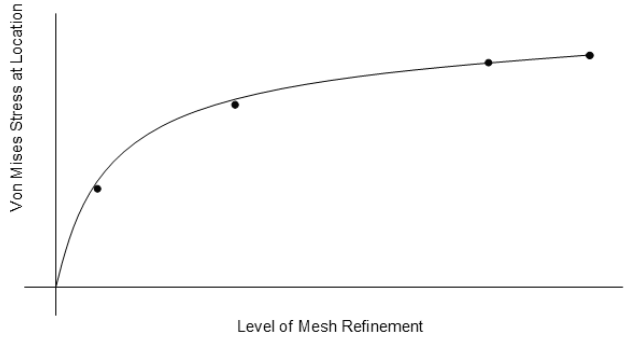
- A "one-sided" error source*
  - Predictions are usually ***lower*** than actual (not higher)
    - Excepting Singularities
  - Nagging feeling because of non-conservative nature
    - Stress is usually underpredicted*
  - Upper bound not determinable
    - Without employing knowledge of materials/loads/element shape functions – discussed later



*Powergraphics results (classic) isn't so one-sided – discussed later

Tet-Pregidious

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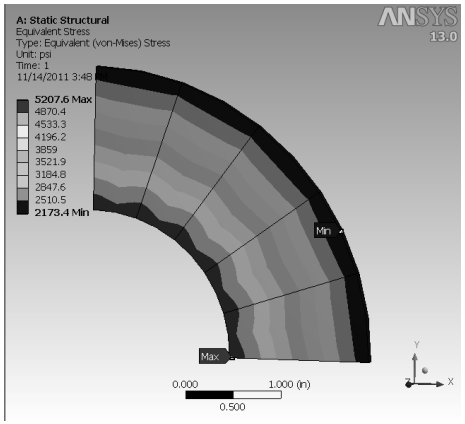
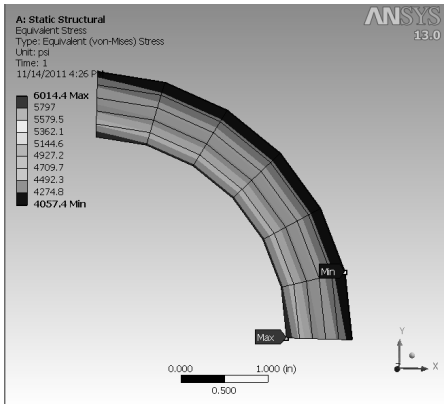
- Bias Against Tetrahedrons (Tet's)
  - Source of grievance?
    - Low order Tets (a.k.a "T4", a.k.a "non-midside noded Tet")
      - Too Stiff in bending / large error with 1 element through thickness
    - 1st Tet's (Berkely 1960's) were high order
    - You'd have to work at it to get ANSYS to create T4's (structural)
  - Tets (10 noded) are Less efficient per DOF
    - Longer solve times
    - Shorter meshing times
    - Added control allows refinement at location of interest
      - **More** efficient than Mapped meshing!
  - Less pleasing to the eye (esp. higher aspect ratios)
  - Stigmatism is receding over last decade

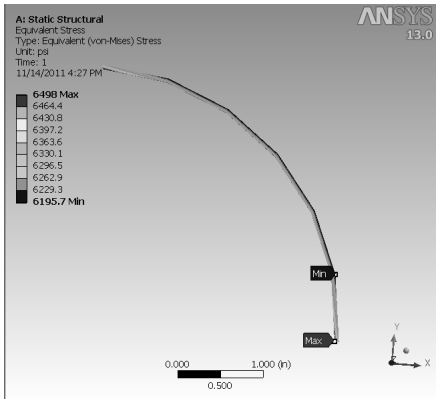
Case Study A

... within Epsilon

- Thick to thin rings with inner pressfit (radial expansion)
  - Stress gradient related to radius^2
- Case Study A, Expansion of Thick/Thin Ring
  - Actually used 5° wedge



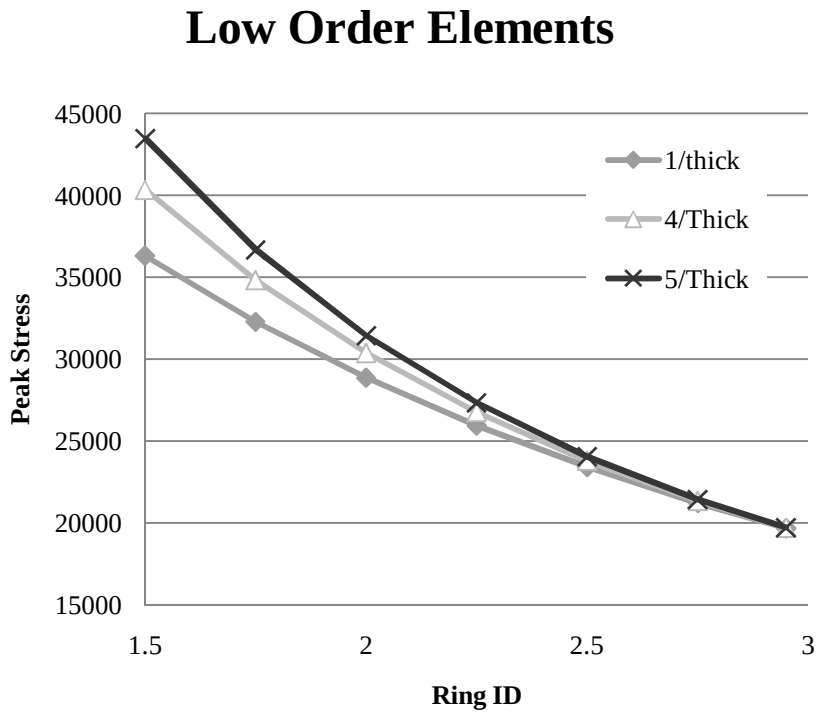


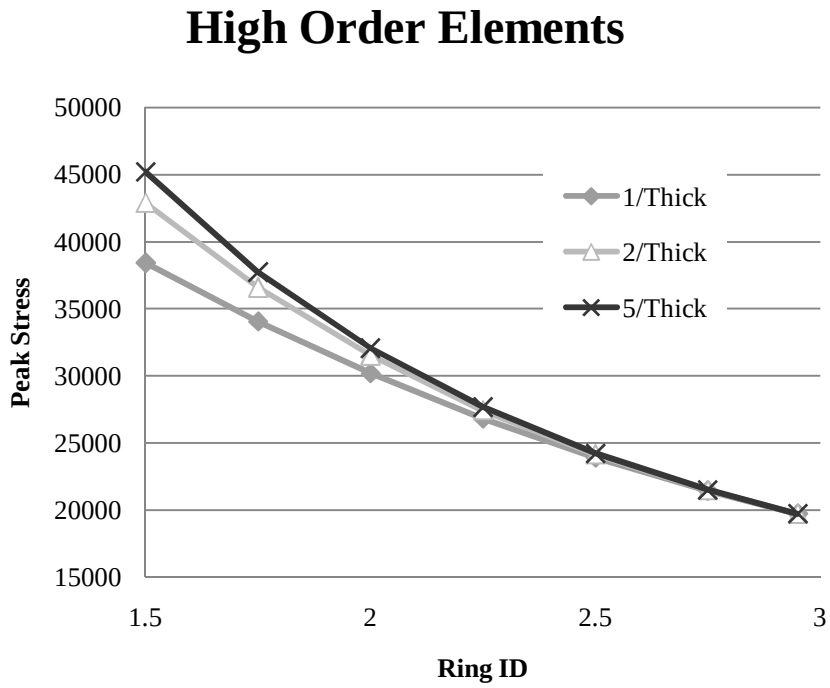


Case Study A

... within Epsilon

- Case Study A
  - Peak Stresses have similar convergence patterns/rate





Case Study A

... within Epsilon

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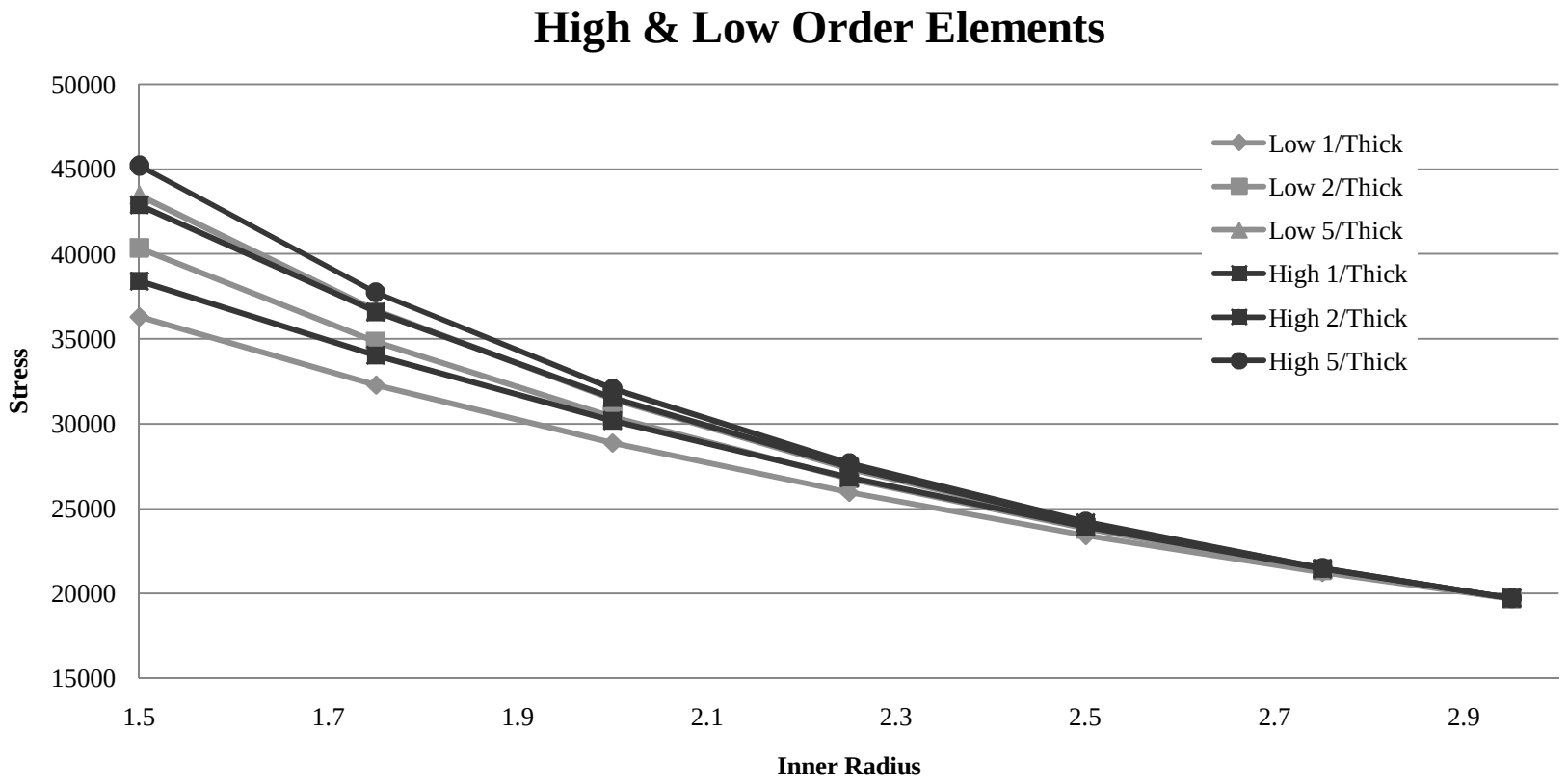
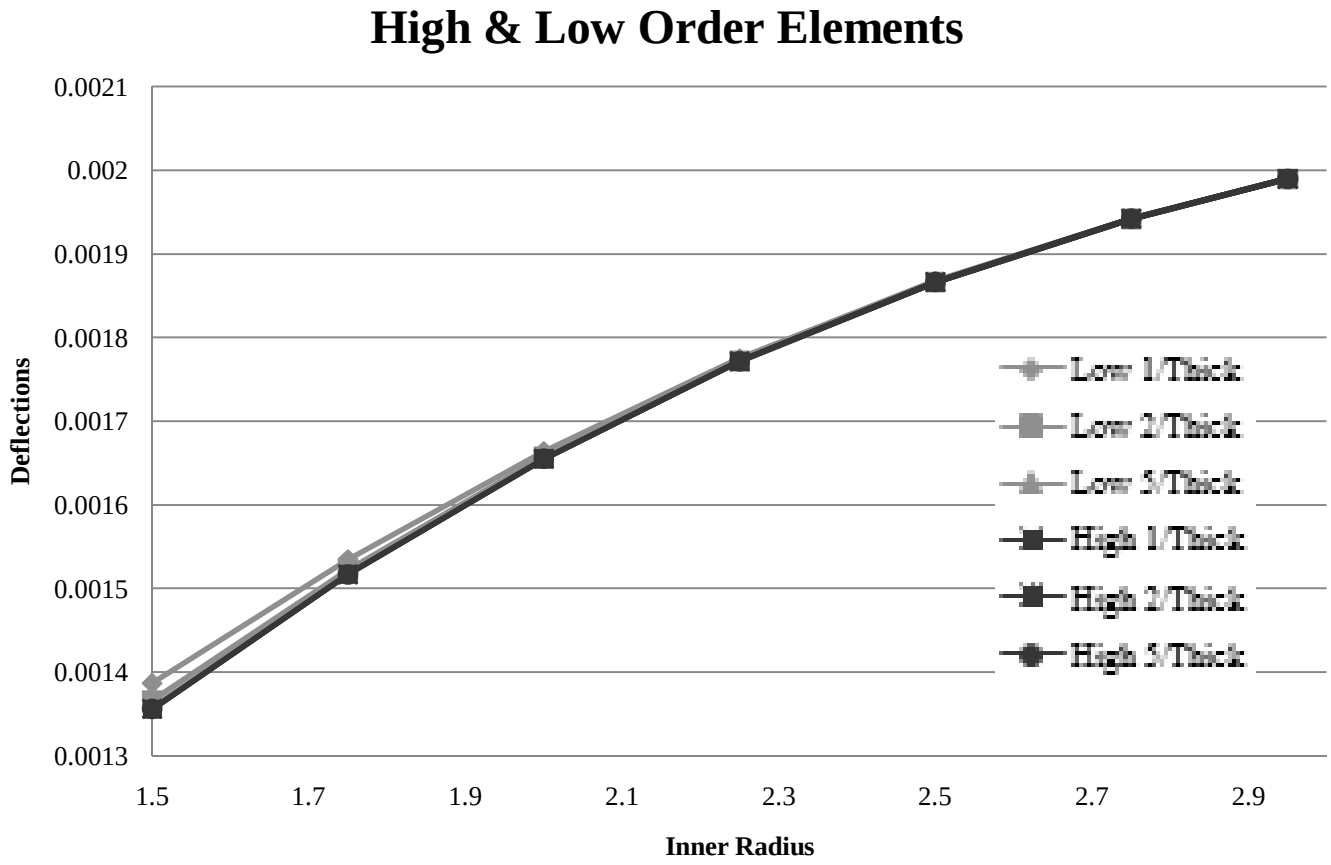
  - Peak Stresses have similar convergence patterns/rate



Case Study A

... within Epsilon

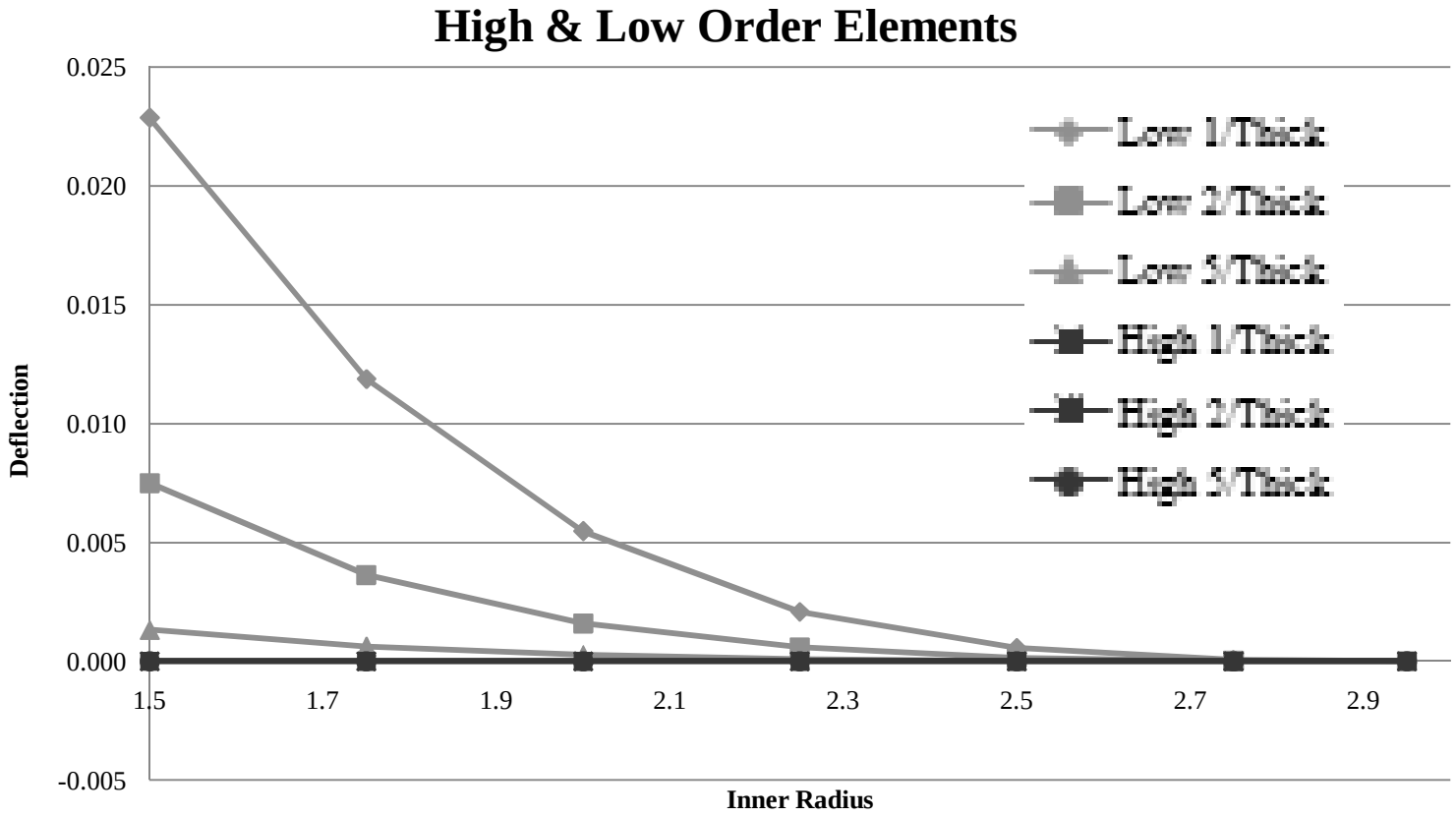
- Case Study A
  - OD Deflections



Case Study A

... within Epsilon

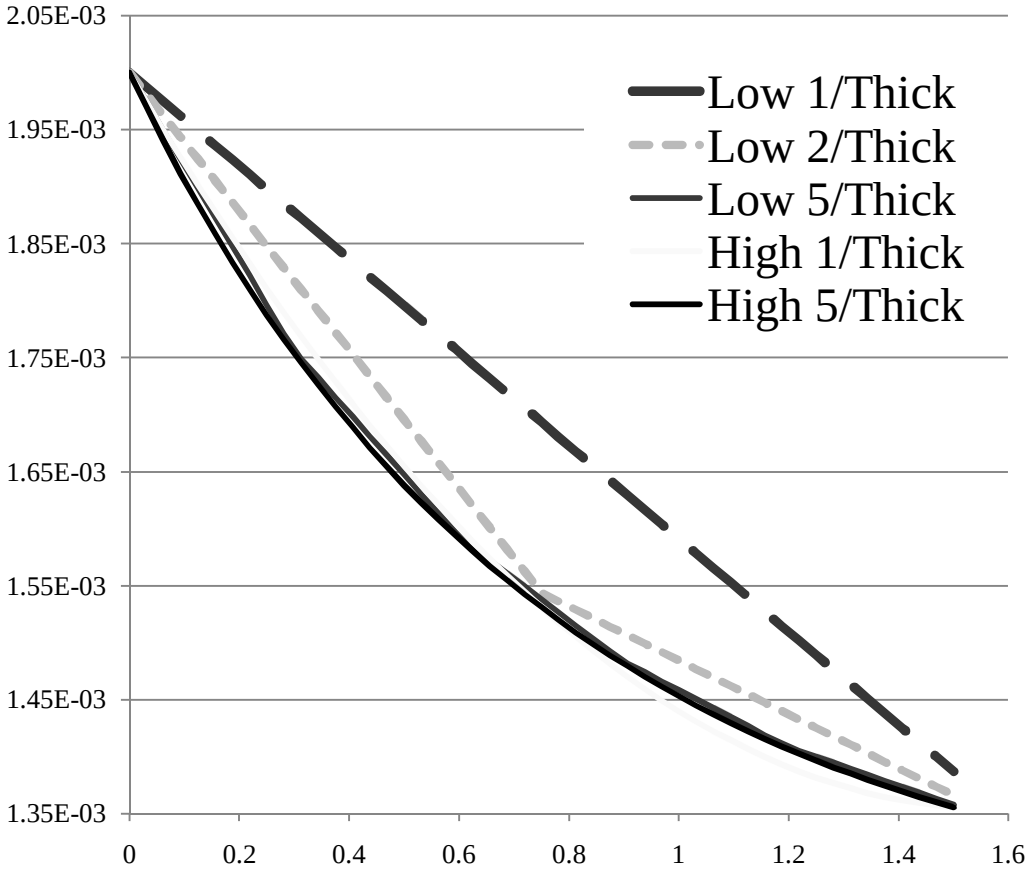
- Case Study A
  - OD Deflections

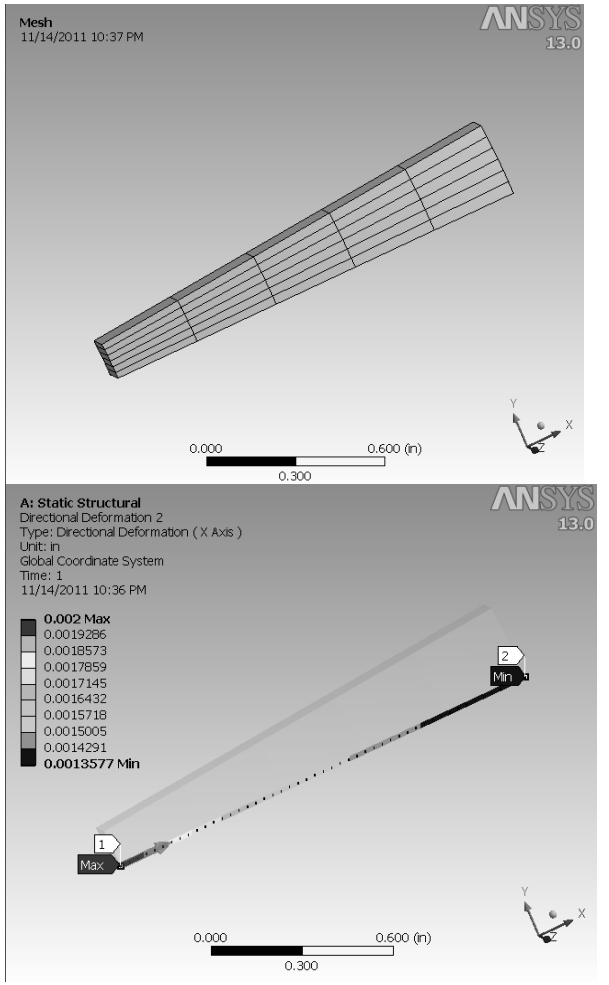


Case Study A

... within Epsilon

- Case Study A
  - Deflections Along Path





Case Study A

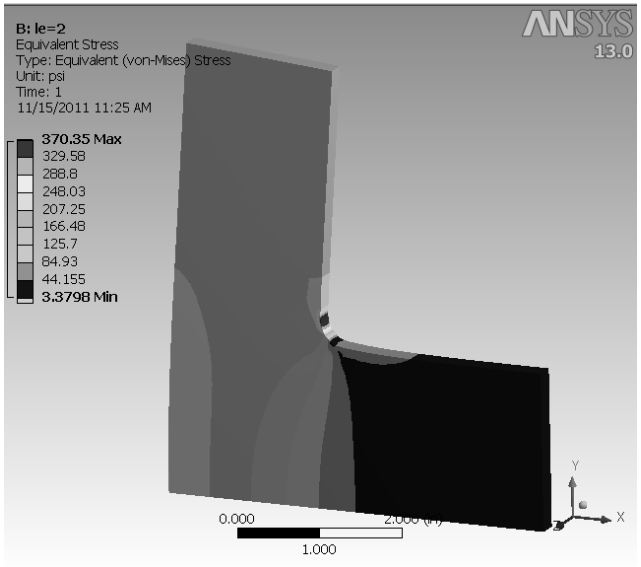
... within Epsilon

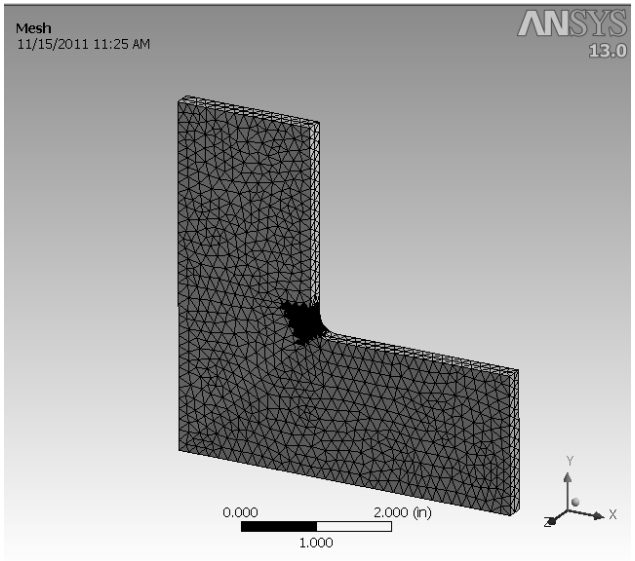
- Case Study A Conclusions
  - Element Stress Gradient
    - Linear for high or low order elements
  - Element Displacement Gradient
    - Linear for low order element
    - 2nd order polynomial for high order element
  - Thin Rings are well approximated with single element through the thickness
    - This extends to beams as well

Mesh Discretization Error

... within Epsilon

- Case Study B
  - Stress along path
  - Node vs. Element (Averaged vs Unaveraged)
  - PRERR (SEPC/SMXB)

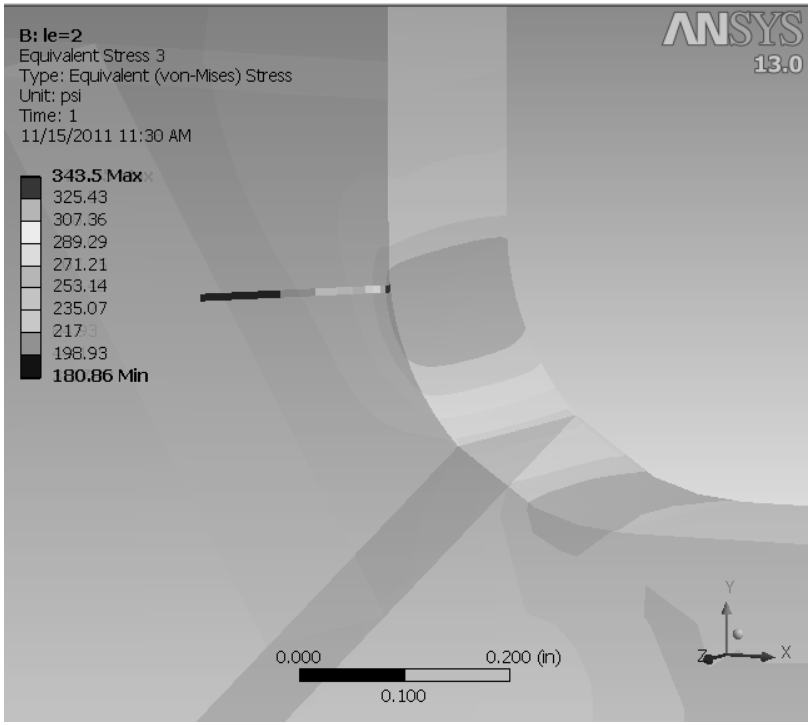


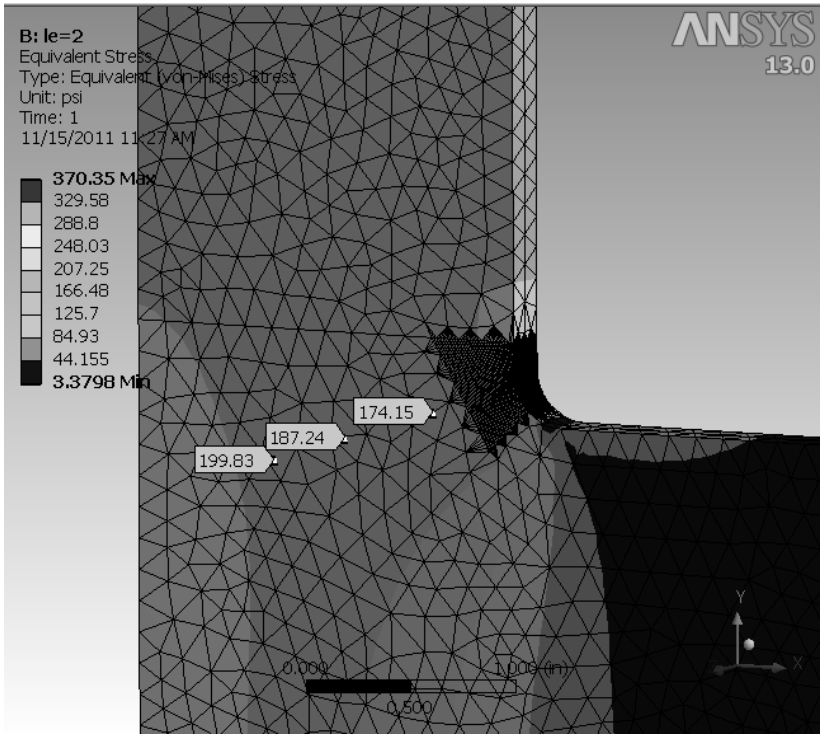


Case Study B

... within Epsilon

- Stress along path
  - Background stress of 180
  - KT =2.0

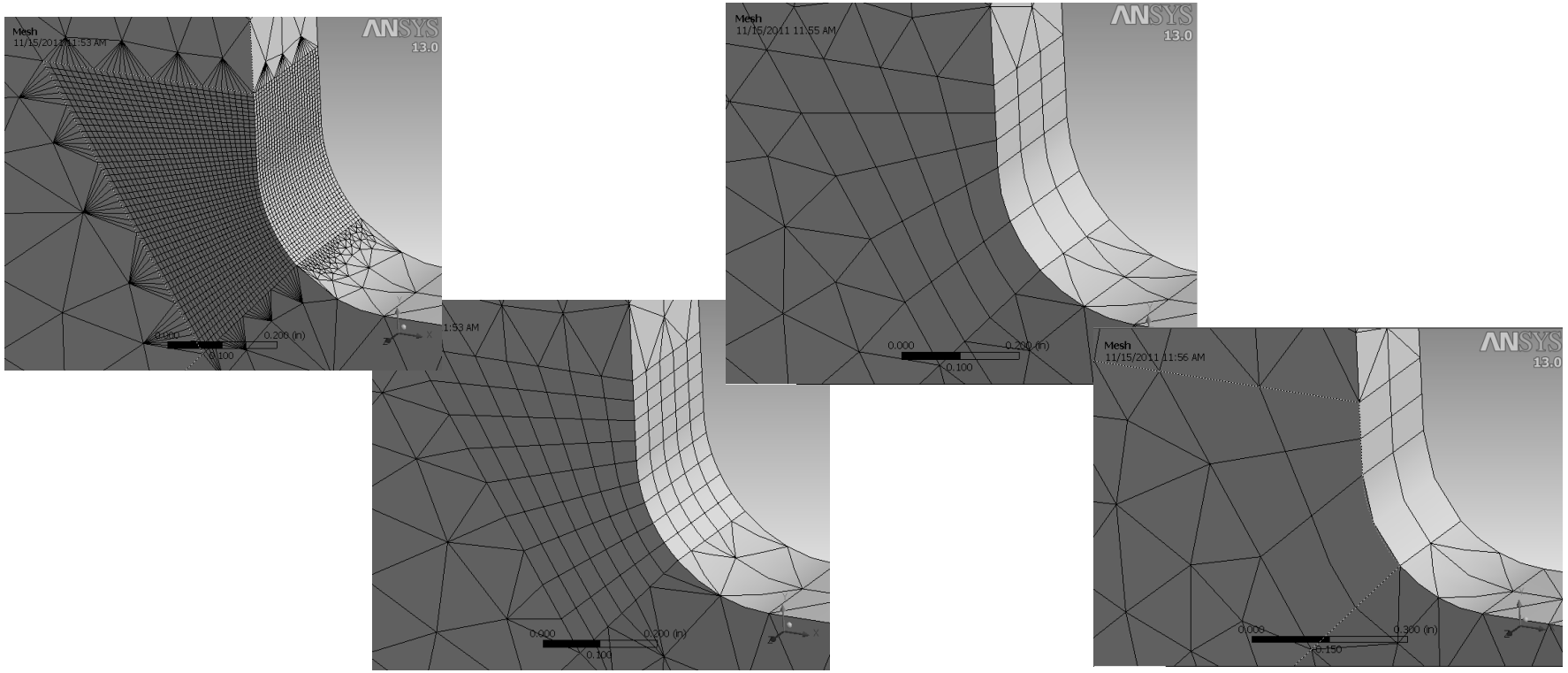




Case Study B

... within Epsilon

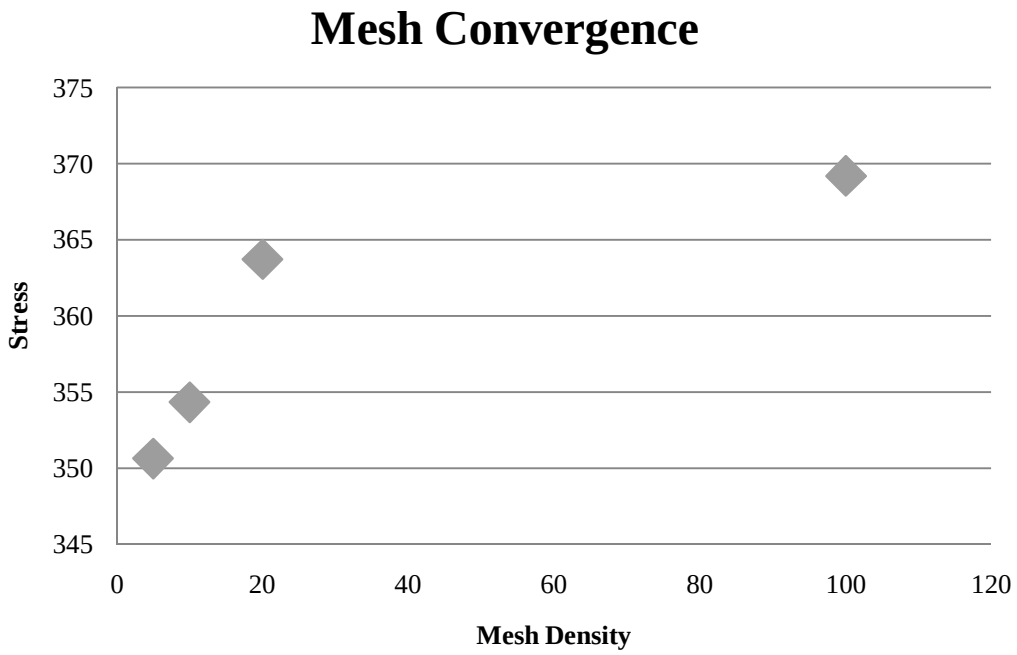
- Stress along path
  - Varying Mesh densities



Case Study B

... within Epsilon

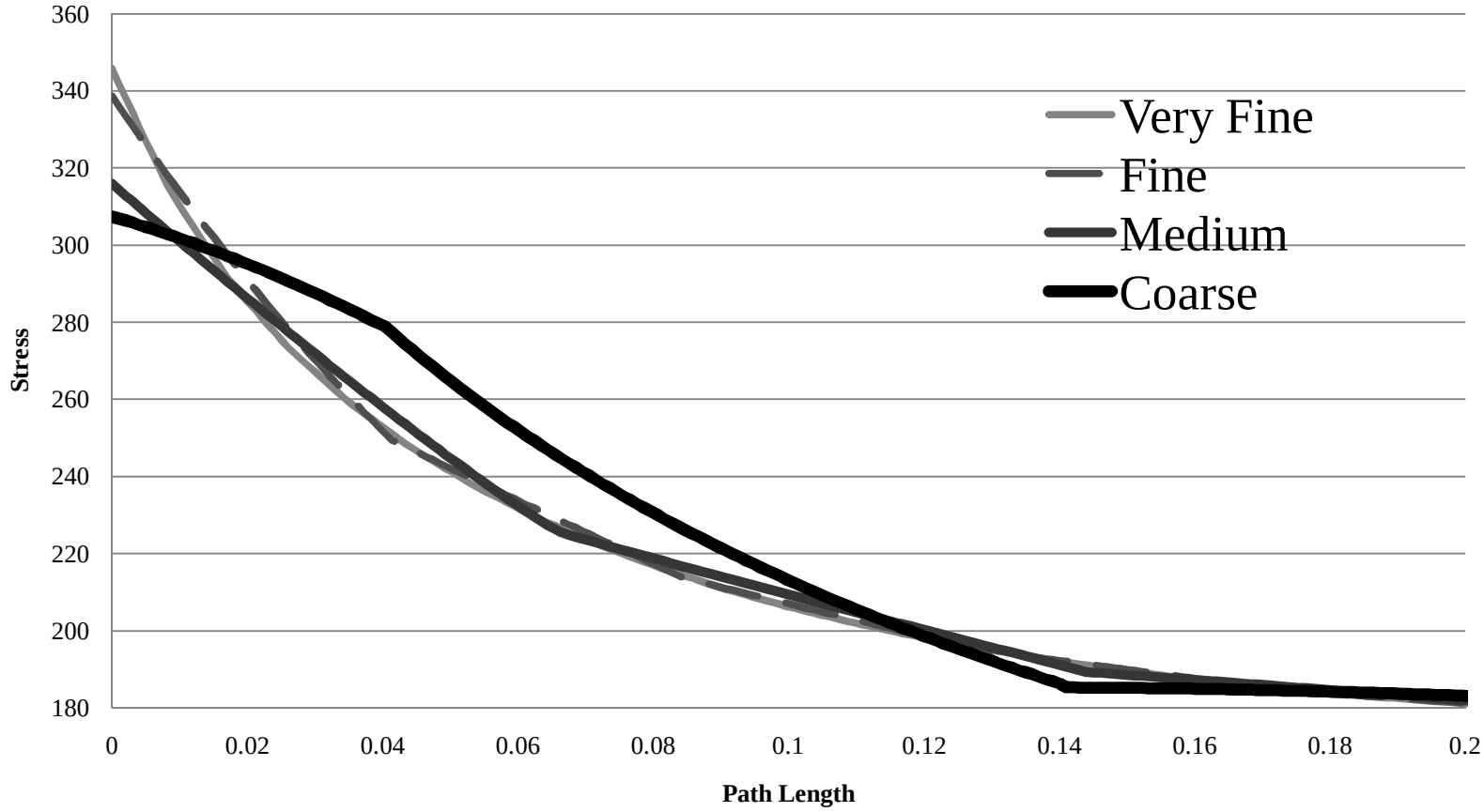
- Peak Stress
  - Varying Mesh densities
    - WB ‘s adaptive mesh refinement automates this task refining only regions of interest (thanks, paul)



Case Study B

... within Epsilon

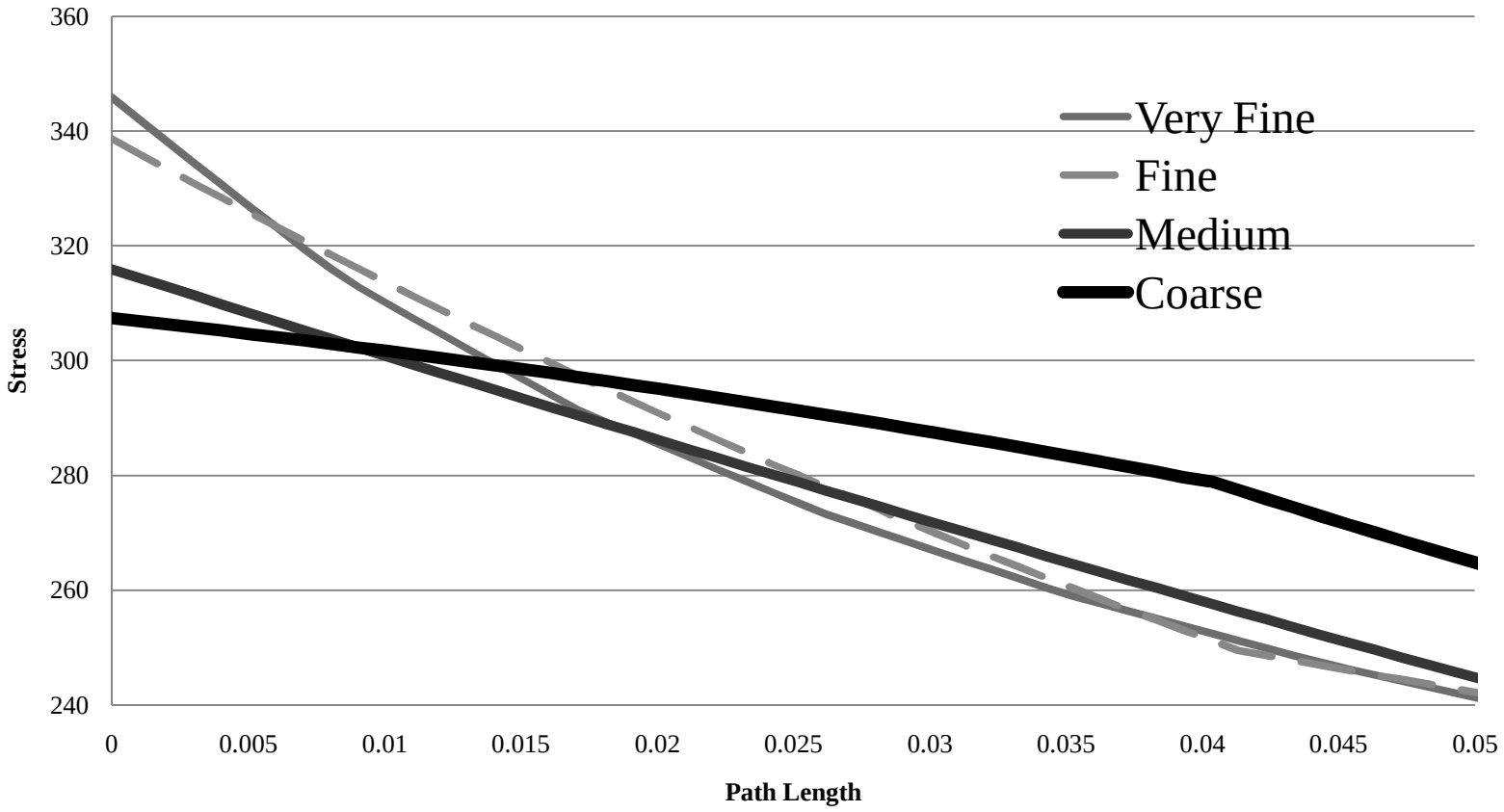
- Stress along path



Case Study B

... within Epsilon

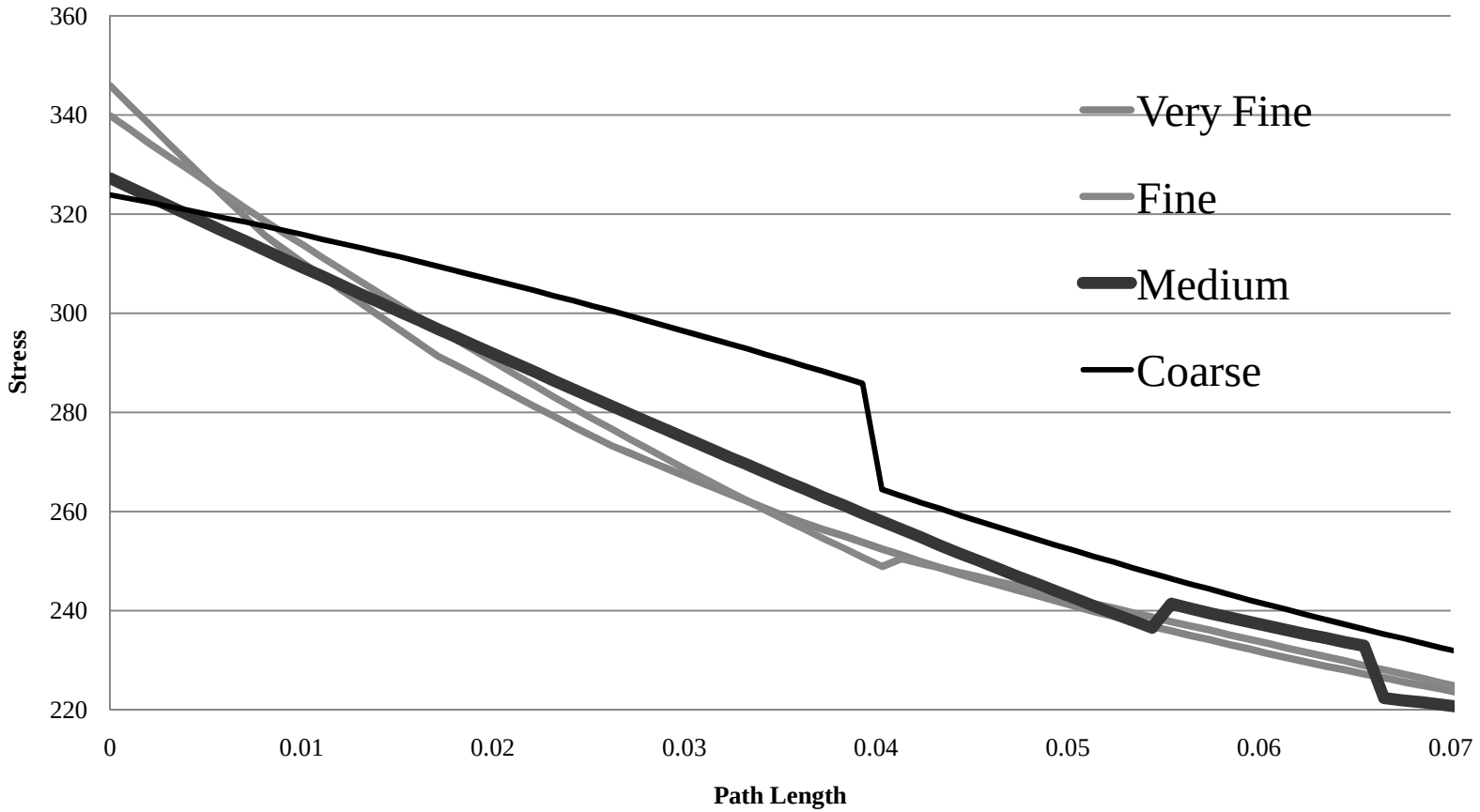
- Stress along path: zoom



Case Study B

... within Epsilon

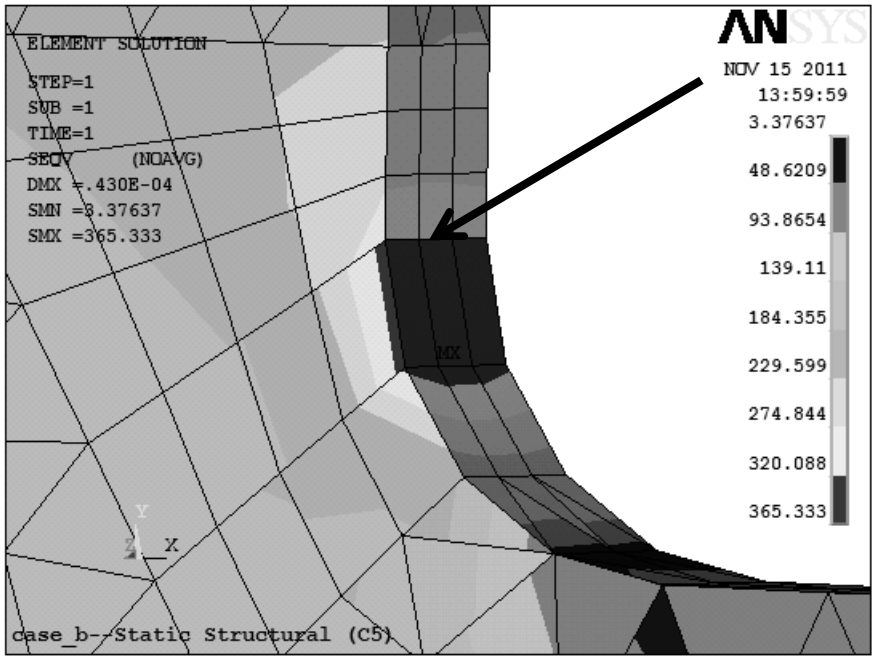
- Stress along path: Unaveraged Results



Case Study B

... within Epsilon

- Case Study B Conclusion:
  - Discontinuity of stress element-to-element relates to degree of mesh discretization error

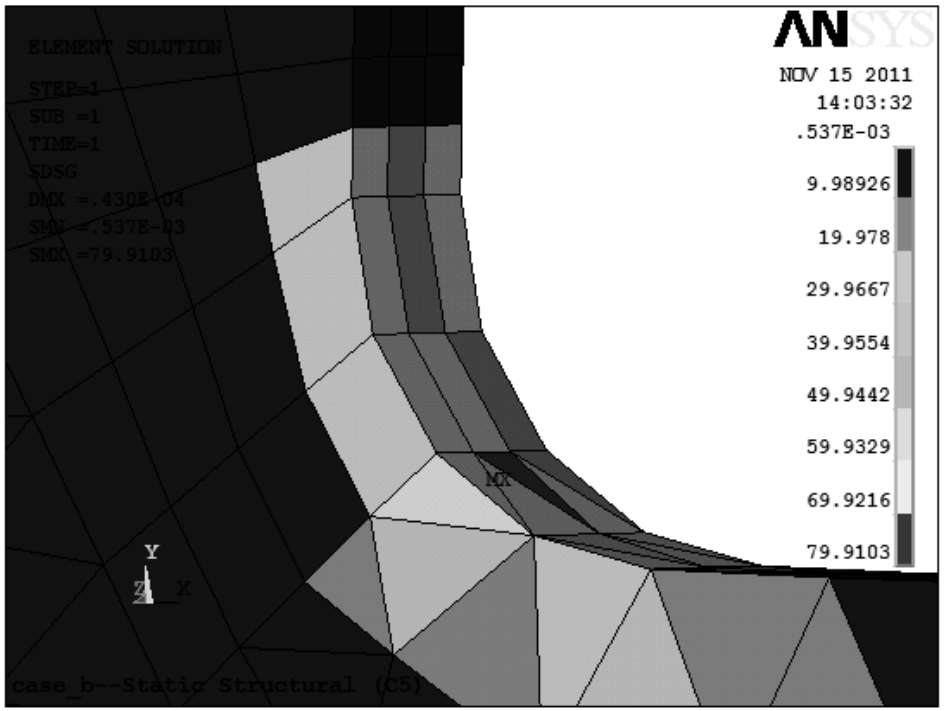


Error Assessment

... within Epsilon

- Discontinuity at element boundaries is key

$$\{\Delta\sigma_n^i\} = \{\sigma_n^a\} - \{\sigma_n^i\}$$ Difference at boundary



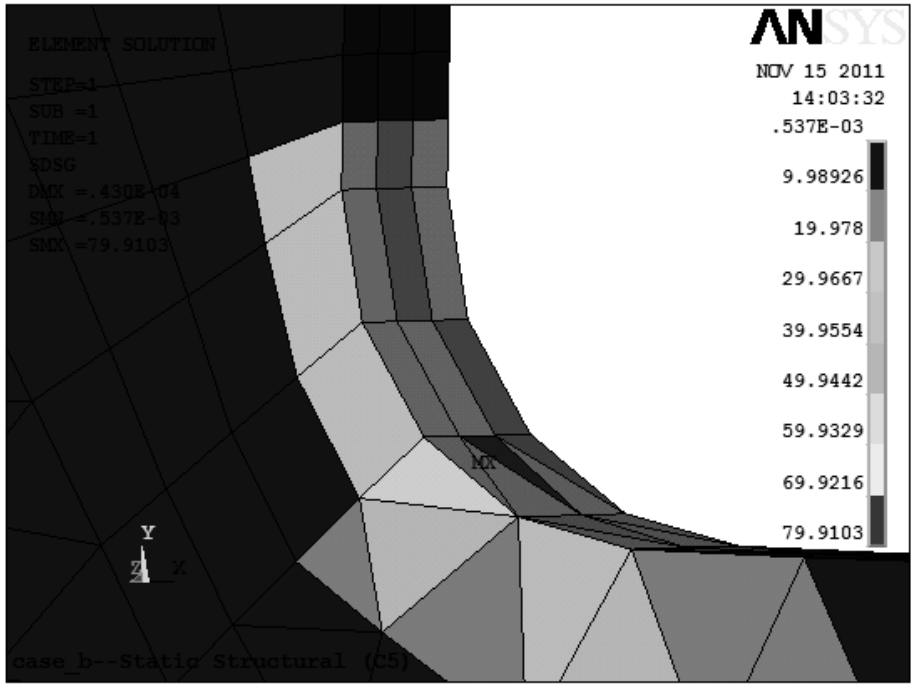
Error Assessment

... within Epsilon

- Discontinuity at element boundaries is key

$$e_i = \frac{1}{2}\int_{vol}\{\Delta\sigma\}^T[D]^{-1}\{\Delta\sigma\}d(vol)$$

- Energy difference per element
- Considers volume/stiffness



Error Assessment

... within Epsilon

- Discontinuity at element boundaries is key

$$e = \sum_{i=1}^{N} e_i$$

- Sum it over the model (selected region)

$$E = 100\left(\frac{e}{U+e}\right)^{\frac{1}{2}}$$

- Normalize it to the whole model energy
  (includes load magnitude)

Yields a single number!

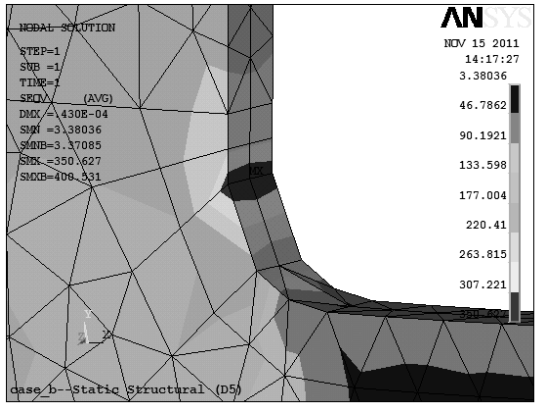
(PRERR, or Percentage error in the energy norm)

Error Assessment

... within Epsilon

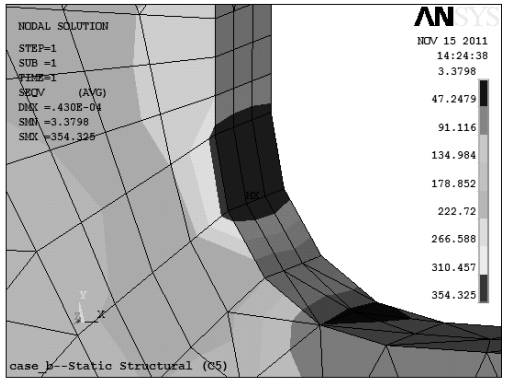
- Percentage error in the energy norm (PRERR)

Coarse

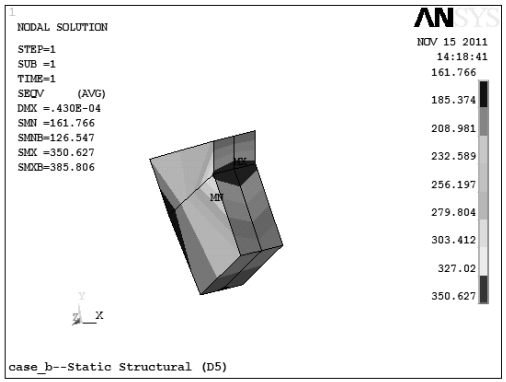


1.59

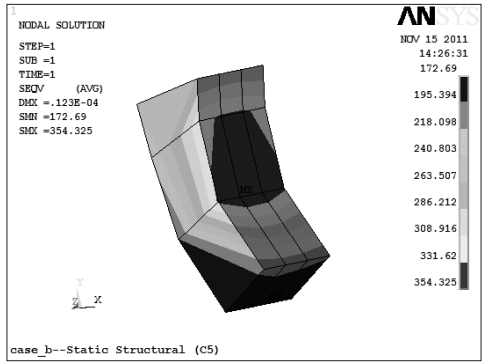
Medium



0.797



8.95



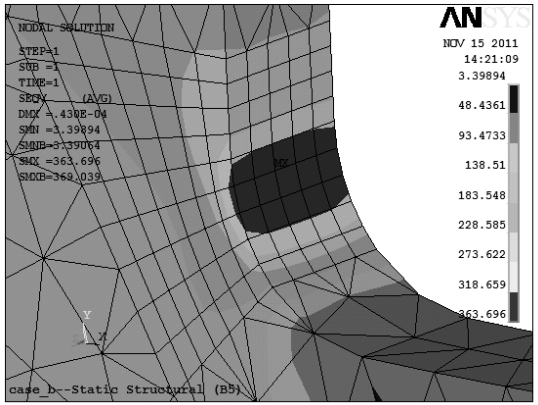
4.0

Error Assessment

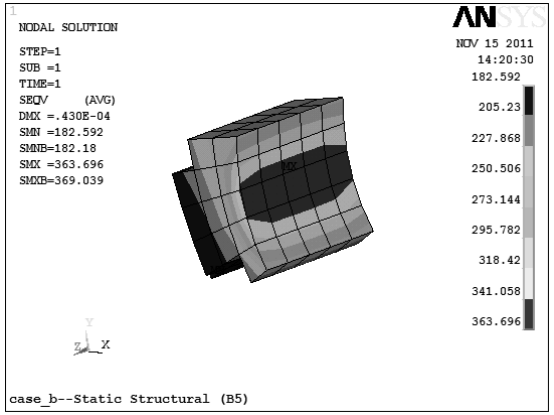
... within Epsilon

- Percentage error in the energy norm (PRERR)

Fine

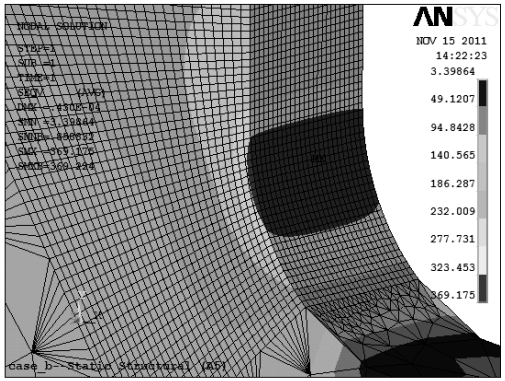


0.56

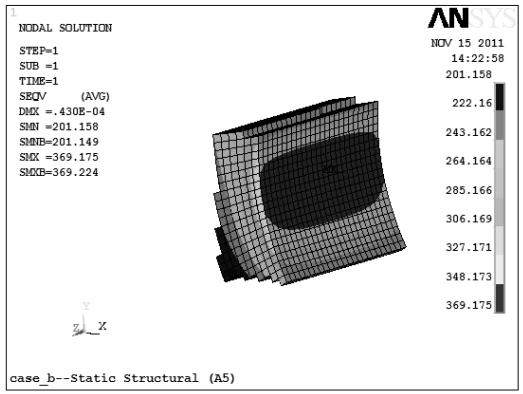


1.13

Very Fine



0.32



0.06

Error Assessment

... within Epsilon

- SMXB
  - Checks all nodes (doesn't necessarily correspond to the MX location!)
  - Only mentioned once in Help Manual!
  - Training Classes refer to it as a "confidence band"…

$$\sigma_j^{mxb} = \max(\sigma_{j,n}^{a} + \Delta\sigma_n)$$

Root Mean Square of:
(avg. value – element value)
for each element sharing node

Average stress from contributing elements (what's plotted)

/EOF